

A Partitioned Finite Element Method for the Structure-Preserving Discretization of Damped Infinite-Dimensional Port-Hamiltonian Systems with Boundary Control

Anass Serhani¹

Denis Matignon¹

Ghislain Haine¹

¹ISAE-Supaero, Toulouse

1 Introduction

- Main Objective
- Definitions and Notations

2 Partitioned Finite Element Method (PFEM)

- Conservative System
- Internal Dissipation
- Boundary Dissipation

3 Conclusion

1 Introduction

- Main Objective
- Definitions and Notations

2 Partitioned Finite Element Method (PFEM)

3 Conclusion

Aim:

Simulate complex open physical systems by ensuring the *conservation of the power balance* for a chosen functional: the **Hamiltonian**.

Aim:

Simulate complex open physical systems by ensuring the *conservation of the power balance* for a chosen functional: the **Hamiltonian**.

- *Finite Element Method*:

- **Complex geometries** are allowed.
- A wide range of **implementation** tools are available.

Aim:

Simulate complex open physical systems by ensuring the *conservation of the power balance* for a chosen functional: the **Hamiltonian**.

- *Finite Element Method*:

- **Complex geometries** are allowed.
- A wide range of **implementation** tools are available.

- *Port-Hamiltonian Systems (PHS)*:

- Model “**energy**” **exchanges** between simpler open subsystems.
- The power balance is *encoded* in a **Stokes-Dirac structure**.

Aim:

Simulate complex open physical systems by ensuring the *conservation of the power balance* for a chosen functional: the **Hamiltonian**.

- *Finite Element Method*:

- **Complex geometries** are allowed.
- A wide range of **implementation** tools are available.

- *Port-Hamiltonian Systems (PHS)*:

- Model “**energy**” **exchanges** between simpler open subsystems.
- The power balance is *encoded* in a **Stokes-Dirac structure**.

- *Partitioned Finite Element Method (PFEM)*:

- It translates the Stokes-Dirac structure into a **Dirac structure**.
- The **discrete Hamiltonian** satisfies the “discrete” power balance.



A structure-preserving Partitioned Finite Element Method for the 2D wave equation
Cardoso-Ribeiro F.L., Matignon D., Lefèvre L.
IFAC-PapersOnLine, vol.51(3), pp.119–124 (2018), LHMNC 2018

Change of paradigm?

Physics

Conservation of mass
Rigid body
“Context and Axioms”
Constant temperature

&
Energy \mathcal{H}

Fourier's law
 $p := mv$
“Definitions and Laws”
Hooke's law

Modelling



Discretization

Change of paradigm?

Physics

Conservation of mass
Rigid body
Constant temperature
“Context and Axioms”

Fourier's law
 $p := mv$
Hooke's law
“Definitions and Laws”

&
Energy \mathcal{H}

Modelling

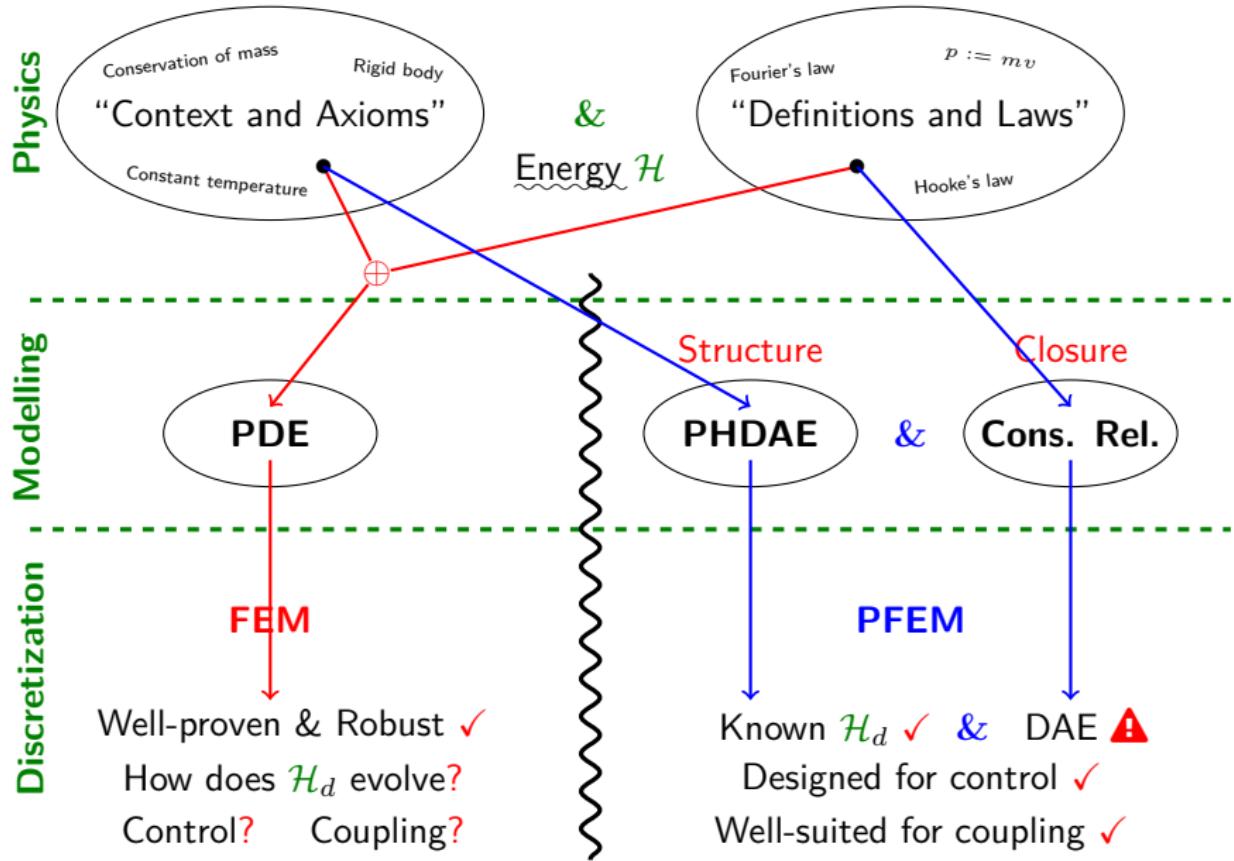
PDE

FEM

Well-proven & Robust ✓
How does \mathcal{H}_d evolve?
Control? Coupling?



Change of paradigm?



1 Introduction

- Main Objective
- Definitions and Notations

2 Partitioned Finite Element Method (PFEM)

3 Conclusion

- The **energy variables** $\vec{\alpha}$ (vector field);
- The **Hamiltonian** $\mathcal{H}(\vec{\alpha}(t))$ (positive functional);

- The **energy variables** $\vec{\alpha}$ (vector field);
- The **Hamiltonian** $\mathcal{H}(\vec{\alpha}(t))$ (positive functional);
- The **co-energy variables** $\vec{e}_{\vec{\alpha}} := \delta_{\vec{\alpha}} \mathcal{H}$ (vector field),
~~~ the variational derivative of  $\mathcal{H}$  w.r.t  $\vec{\alpha}$ ;

- The **energy variables**  $\vec{\alpha}$  (vector field);
- The **Hamiltonian**  $\mathcal{H}(\vec{\alpha}(t))$  (positive functional);
- The **co-energy variables**  $\vec{e}_\alpha := \delta_{\vec{\alpha}} \mathcal{H}$  (vector field),  
~~ the variational derivative of  $\mathcal{H}$  w.r.t  $\vec{\alpha}$ ;
- The **structure operator**  $J$  (linear and *formally skew-symmetric*);
- The **resistive/dissipative operator**  $R$  (linear and *positive*);

- The **energy variables**  $\vec{\alpha}$  (vector field);
- The **Hamiltonian**  $\mathcal{H}(\vec{\alpha}(t))$  (positive functional);
- The **co-energy variables**  $\vec{e}_\alpha := \delta_{\vec{\alpha}} \mathcal{H}$  (vector field),  
     $\leadsto$  the variational derivative of  $\mathcal{H}$  w.r.t  $\vec{\alpha}$ ;
- The **structure operator**  $J$  (linear and *formally skew-symmetric*);
- The **resistive/dissipative operator**  $R$  (linear and *positive*);
- The **control operator**  $B$  (linear);
- The **input**  $u$  and the **collocated output**  $y$  (boundary scalar fields);

- The **energy variables**  $\vec{\alpha}$  (vector field);
- The **Hamiltonian**  $\mathcal{H}(\vec{\alpha}(t))$  (positive functional);
- The **co-energy variables**  $\vec{e}_{\vec{\alpha}} := \delta_{\vec{\alpha}} \mathcal{H}$  (vector field),  
~~ the variational derivative of  $\mathcal{H}$  w.r.t  $\vec{\alpha}$ ;
- The **structure operator**  $J$  (linear and *formally skew-symmetric*);
- The **resistive/dissipative operator**  $R$  (linear and *positive*);
- The **control operator**  $B$  (linear);
- The **input  $u$**  and the **collocated output  $y$**  (boundary scalar fields);
- The **dynamical system**:

$$\begin{cases} \partial_t \vec{\alpha}(t) = (J - R) \vec{e}_{\vec{\alpha}}(t) + B \mathbf{u}(t), \\ \mathbf{y}(t) = B^* \vec{e}_{\vec{\alpha}}(t). \end{cases}$$

- The **energy variables**  $\vec{\alpha}$  (vector field);
- The **Hamiltonian**  $\mathcal{H}(\vec{\alpha}(t))$  (positive functional);
- The **co-energy variables**  $\vec{e}_{\vec{\alpha}} := \delta_{\vec{\alpha}} \mathcal{H}$  (vector field),  
~~ the variational derivative of  $\mathcal{H}$  w.r.t  $\vec{\alpha}$ ;
- The **structure operator**  $J$  (linear and *formally skew-symmetric*);
- The **resistive/dissipative operator**  $R$  (linear and *positive*);
- The **control operator**  $B$  (linear);
- The **input  $u$**  and the **collocated output  $y$**  (boundary scalar fields);
- The **dynamical system**:

$$\begin{cases} \partial_t \vec{\alpha}(t) = (J - R) \vec{e}_{\vec{\alpha}}(t) + B \mathbf{u}(t), \\ \mathbf{y}(t) = B^* \vec{e}_{\vec{\alpha}}(t). \end{cases}$$

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}(t)) = - \langle R \vec{e}_{\vec{\alpha}}(t), \vec{e}_{\vec{\alpha}}(t) \rangle_J + \langle \mathbf{u}(t), \mathbf{y}(t) \rangle_B \leq \langle \mathbf{u}(t), \mathbf{y}(t) \rangle_B.$$

⚠ Although the **underlying geometry** is well-determined with the above equality, **constitutive relations** between  $\vec{\alpha}$  and  $\vec{e}_{\vec{\alpha}}$  are also needed to solve the system!

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_{\alpha}, \vec{e}_R, u)^\top$ ;

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_{\vec{\alpha}}, \vec{e}_R, \mathbf{u})^\top$ ;
- The **flow space**  $\mathcal{F} := \mathcal{E}'$  and  $\vec{f} := (\partial_t \vec{\alpha}, \vec{f}_R, -\mathbf{y})^\top$ ;

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_{\alpha}, \vec{e}_R, u)^\top$ ;
- The **flow space**  $\mathcal{F} := \mathcal{E}'$  and  $\vec{f} := (\partial_t \vec{\alpha}, \vec{f}_R, -y)^\top$ ;
- The **input–output structure operator**  $\mathcal{J} := \begin{pmatrix} J & -I & B \\ I & 0 & 0 \\ -B^* & 0 & 0 \end{pmatrix}$ ;

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_{\alpha}, \vec{e}_R, u)^\top$ ;
- The **flow space**  $\mathcal{F} := \mathcal{E}'$  and  $\vec{f} := (\partial_t \vec{\alpha}, \vec{f}_R, -y)^\top$ ;
- The **input–output structure operator**  $\mathcal{J} := \begin{pmatrix} J & -I & B \\ I & 0 & 0 \\ -B^* & 0 & 0 \end{pmatrix}$ ;
- The **Bond space**  $\mathcal{B} := \mathcal{F} \times \mathcal{E}$ , with symmetrized bilinear product:

$$\left[ \begin{pmatrix} \vec{f}^1 \\ \vec{e}^1 \end{pmatrix}, \begin{pmatrix} \vec{f}^2 \\ \vec{e}^2 \end{pmatrix} \right]_{\mathcal{B}} := \left\langle \vec{f}^1, \vec{e}^2 \right\rangle_{\mathcal{F}, \mathcal{E}} + \left\langle \vec{f}^2, \vec{e}^1 \right\rangle_{\mathcal{F}, \mathcal{E}};$$

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_{\alpha}, \vec{e}_R, u)^\top$ ;
- The **flow space**  $\mathcal{F} := \mathcal{E}'$  and  $\vec{f} := (\partial_t \vec{\alpha}, \vec{f}_R, -y)^\top$ ;
- The **input–output structure operator**  $\mathcal{J} := \begin{pmatrix} J & -I & B \\ I & 0 & 0 \\ -B^* & 0 & 0 \end{pmatrix}$ ;
- The **Bond space**  $\mathcal{B} := \mathcal{F} \times \mathcal{E}$ , with symmetrized bilinear product:

$$\left[ \begin{pmatrix} \vec{f}^1 \\ \vec{e}^1 \end{pmatrix}, \begin{pmatrix} \vec{f}^2 \\ \vec{e}^2 \end{pmatrix} \right]_{\mathcal{B}} := \left\langle \vec{f}^1, \vec{e}^2 \right\rangle_{\mathcal{F}, \mathcal{E}} + \left\langle \vec{f}^2, \vec{e}^1 \right\rangle_{\mathcal{F}, \mathcal{E}};$$

- The **Dirac structure**  $\mathcal{D} := \text{Graph}(\mathcal{J}) \subset \mathcal{B}$ , i.e.  $\mathcal{D}^{[\perp]} = \mathcal{D}$  with:

$$\mathcal{D}^{[\perp]} := \left\{ \begin{pmatrix} \vec{f} \\ \vec{e} \end{pmatrix} \in \mathcal{B} \mid \left[ \begin{pmatrix} \vec{f} \\ \vec{e} \end{pmatrix}, \begin{pmatrix} \widetilde{f} \\ \widetilde{e} \end{pmatrix} \right]_{\mathcal{B}} = 0, \quad \forall \begin{pmatrix} \widetilde{f} \\ \widetilde{e} \end{pmatrix} \in \mathcal{D} \right\}.$$

⚠ Hypotheses on  $J$  and  $B$  are needed for  $\mathcal{D}$  to be a (Stokes-)Dirac structure!

# Associated (Stokes-)Dirac structures

- The **effort space**  $\mathcal{E}$  (Hilbert space) and  $\vec{e} := (\vec{e}_\alpha, \vec{e}_R, u)^\top$ ;
- The **flow space**  $\mathcal{F} := \mathcal{E}'$  and  $\vec{f} := (\partial_t \vec{\alpha}, \vec{f}_R, -y)^\top$ ;
- The **input-output structure operator**  $\mathcal{J} := \begin{pmatrix} J & -I & B \\ I & 0 & 0 \\ -B^* & 0 & 0 \end{pmatrix}$ ;
- The **Bond space**  $\mathcal{B} := \mathcal{F} \times \mathcal{E}$ , with symmetrized bilinear product:

$$\left[ \begin{pmatrix} \vec{f}^1 \\ \vec{e}^1 \end{pmatrix}, \begin{pmatrix} \vec{f}^2 \\ \vec{e}^2 \end{pmatrix} \right]_{\mathcal{B}} := \left\langle \vec{f}^1, \vec{e}^2 \right\rangle_{\mathcal{F}, \mathcal{E}} + \left\langle \vec{f}^2, \vec{e}^1 \right\rangle_{\mathcal{F}, \mathcal{E}};$$

- The **Dirac structure**  $\mathcal{D} := \text{Graph}(\mathcal{J}) \subset \mathcal{B}$ , i.e.  $\mathcal{D}^{[\perp]} = \mathcal{D}$  with:

$$\mathcal{D}^{[\perp]} := \left\{ \begin{pmatrix} \vec{f} \\ \vec{e} \end{pmatrix} \in \mathcal{B} \mid \left[ \begin{pmatrix} \vec{f} \\ \vec{e} \end{pmatrix}, \begin{pmatrix} \widetilde{f} \\ \widetilde{e} \end{pmatrix} \right]_{\mathcal{B}} = 0, \quad \forall \begin{pmatrix} \widetilde{f} \\ \widetilde{e} \end{pmatrix} \in \mathcal{D} \right\}.$$

- The **dissipative constitutive relation**  $\vec{e}_R = R \vec{f}_R$ ;

**⚠** Hypotheses on  $J$  and  $B$  are needed for  $\mathcal{D}$  to be a (Stokes-)Dirac structure!

$$\left\langle \vec{f}(t), \vec{e}(t) \right\rangle_{\mathcal{F}, \mathcal{E}} = 0, \quad \forall \left( \vec{f}(t), \vec{e}(t) \right) \in \mathcal{D}, \quad \forall t \geq 0.$$

$$\left\langle \vec{f}(t), \vec{e}(t) \right\rangle_{\mathcal{F}, \mathcal{E}} = 0, \quad \forall \left( \vec{f}(t), \vec{e}(t) \right) \in \mathcal{D}, \quad \forall t \geq 0.$$

Let  $\left( \partial_t \vec{\alpha}, \vec{f}_R, -\vec{y}, \vec{e}_{\vec{\alpha}}, \vec{e}_R, \vec{u} \right)^\top$  be in  $\mathcal{D}$ .

Adding  $\vec{e}_R = R \vec{f}_R$ : **the lossy power balance is satisfied!**

$$\left\langle \vec{f}(t), \vec{e}(t) \right\rangle_{\mathcal{F}, \mathcal{E}} = 0, \quad \forall \left( \vec{f}(t), \vec{e}(t) \right) \in \mathcal{D}, \quad \forall t \geq 0.$$

Let  $\left( \partial_t \vec{\alpha}, \vec{f}_R, -\vec{y}, \vec{e}_{\vec{\alpha}}, \vec{e}_R, \vec{u} \right)^\top$  be in  $\mathcal{D}$ .

Adding  $\vec{e}_R = R \vec{f}_R$ : **the lossy power balance is satisfied!**

⇒ Flow/effort representation **generalizes** the above state representation with  $J - R$ , **and PFEM appears to be very well-suited to it!**

PHS + DAE = **PHDAE**.

$$\left\langle \vec{f}(t), \vec{e}(t) \right\rangle_{\mathcal{F}, \mathcal{E}} = 0, \quad \forall \left( \vec{f}(t), \vec{e}(t) \right) \in \mathcal{D}, \quad \forall t \geq 0.$$

Let  $\left( \partial_t \vec{\alpha}, \vec{f}_R, -\vec{y}, \vec{e}_{\vec{\alpha}}, \vec{e}_R, \vec{u} \right)^\top$  be in  $\mathcal{D}$ .

Adding  $\vec{e}_R = R \vec{f}_R$ : **the lossy power balance is satisfied!**

⇒ Flow/effort representation **generalizes** the above state representation with  $J - R$ , **and PFEM appears to be very well-suited to it!**

$$\text{PHS} + \text{DAE} = \text{PHDAE}.$$

## Main result

PFEM gives rise to a **finite-dimensional Dirac structure** *containing* a **discrete version of the (lossy) power balance** for the **discrete Hamiltonian**.

## 1 Introduction

## 2 Partitioned Finite Element Method (PFEM)

- Conservative System
- Internal Dissipation
- Boundary Dissipation

## 3 Conclusion

Deflection  $w$  of a 2D-membrane, *boundary deflection velocity as control*.

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{\mathbf{T}}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;
- $\vec{e}_q := \delta_{\vec{\alpha}_q} \mathcal{H} = \bar{\bar{T}} \cdot \vec{\alpha}_q$  the *stress*;
- $e_p := \delta_{\alpha_p} \mathcal{H} = \frac{\alpha_p}{\rho}$  the *deflection velocity* and  $\mathbf{u} := \mathbf{e}_p$ ;

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;
- $\vec{e}_q := \delta_{\vec{\alpha}_q} \mathcal{H} = \bar{\bar{T}} \cdot \vec{\alpha}_q$  the *stress*;
- $e_p := \delta_{\alpha_p} \mathcal{H} = \frac{\alpha_p}{\rho}$  the *deflection velocity* and  $\mathbf{u} := e_p$ ;
- $\mathbf{y} := \vec{e}_q \cdot \vec{n}$  the output *normal stress*.

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;
- $\vec{e}_q := \delta_{\vec{\alpha}_q} \mathcal{H} = \bar{\bar{T}} \cdot \vec{\alpha}_q$  the *stress*;
- $e_p := \delta_{\alpha_p} \mathcal{H} = \frac{\alpha_p}{\rho}$  the *deflection velocity* and  $\mathbf{u} := e_p$ ;
- $\mathbf{y} := \vec{e}_q \cdot \vec{n}$  the output *normal stress*.

$$\left\{ \begin{array}{l} \rho \partial_{tt}^2 w = \text{div} \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right), \\ \mathbf{u} = \partial_t w, \\ \mathbf{y} = \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right) \cdot \vec{n}. \end{array} \right.$$

# Conservative System: Wave as PDAE

**Deflection  $w$  of a 2D-membrane, boundary deflection velocity as control.**

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;
- $\vec{e}_q := \delta_{\vec{\alpha}_q} \mathcal{H} = \bar{\bar{T}} \cdot \vec{\alpha}_q$  the *stress*;
- $e_p := \delta_{\alpha_p} \mathcal{H} = \frac{\alpha_p}{\rho}$  the *deflection velocity* and  $\mathbf{u} := e_p$ ;
- $\mathbf{y} := \vec{e}_q \cdot \vec{n}$  the output *normal stress*.

$$\left\{ \begin{array}{l} \rho \partial_{tt}^2 w = \text{div} \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right), \\ \mathbf{u} = \partial_t w, \\ \mathbf{y} = \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right) \cdot \vec{n}. \end{array} \right. \Leftrightarrow \left\{ \begin{array}{l} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(e_p), \\ \partial_t \alpha_p = \text{div}(\vec{e}_q), \end{array} \right. \text{ & } \left\{ \begin{array}{l} \mathbf{u} = e_p, \\ \mathbf{y} = \vec{e}_q \cdot \vec{n}. \end{array} \right.$$

Deflection  $w$  of a 2D-membrane, *boundary deflection velocity* as control.

Its total energy is given by the sum of the potential & kinetic energies:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{T}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

- $\rho$  the *mass density* of the medium and  $\bar{\bar{T}}$  the *Young modulus tensor*;
- $\alpha_p := \rho \partial_t w$  the *linear momentum* and  $\vec{\alpha}_q := \overrightarrow{\text{grad}}(w)$  the *strain*;
- $\vec{e}_q := \delta_{\vec{\alpha}_q} \mathcal{H} = \bar{\bar{T}} \cdot \vec{\alpha}_q$  the *stress*;
- $e_p := \delta_{\alpha_p} \mathcal{H} = \frac{\alpha_p}{\rho}$  the *deflection velocity* and  $\mathbf{u} := e_p$ ;
- $\mathbf{y} := \vec{e}_q \cdot \vec{n}$  the output *normal stress*.

$$\left\{ \begin{array}{l} \rho \partial_{tt}^2 w = \text{div} \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right), \\ \mathbf{u} = \partial_t w, \\ \mathbf{y} = \left( \bar{\bar{T}} \cdot \overrightarrow{\text{grad}}(w) \right) \cdot \vec{n}. \end{array} \right. \Leftrightarrow \left\{ \begin{array}{l} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(e_p), \\ \partial_t \alpha_p = \text{div}(\vec{e}_q), \end{array} \right. \text{ & } \left\{ \begin{array}{l} \mathbf{u} = e_p, \\ \mathbf{y} = \vec{e}_q \cdot \vec{n}. \end{array} \right.$$

## Lossless Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}_q, \alpha_p) = \langle \mathbf{y}, \mathbf{u} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

The strategy follows:

The strategy follows:

- 1 Write the **weak formulation**;

The strategy follows:

- 1 Write the **weak formulation**;
- 2 Apply an accurate **Stokes (Green) identity** (such that  $\mathbf{u}$  “appears”);

The strategy follows:

- 1 Write the **weak formulation**;
- 2 Apply an accurate **Stokes (Green) identity** (such that  $u$  “appears”);
- 3 Project on a finite-dimensional space thanks to **FEM**.

The strategy follows:

- 1 Write the **weak formulation**;
- 2 Apply an accurate **Stokes (Green) identity** (such that  $\mathbf{u}$  “appears”);
- 3 Project on a finite-dimensional space thanks to **FEM**.

For all test functions  $\vec{v}_q$ ,  $v_p$  and  $v_\partial$  (smooth enough):

$$\left\{ \begin{array}{l} \langle \partial_t \vec{\alpha}_q, \vec{v}_q \rangle_{\mathbf{L}^2} = \langle \overrightarrow{\mathbf{grad}}(\mathbf{e}_p), \vec{v}_q \rangle_{\mathbf{L}^2}, \\ \langle \partial_t \alpha_p, v_p \rangle_{L^2} = \langle \operatorname{div}(\vec{\mathbf{e}}_q), v_p \rangle_{L^2}, \\ \langle \mathbf{y}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} = \langle \vec{\mathbf{e}}_q \cdot \vec{\mathbf{n}}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}. \end{array} \right.$$

The strategy follows:

- 1 Write the **weak formulation**;
- 2 Apply an accurate **Stokes (Green) identity** (such that  $\mathbf{u}$  “appears”);
- 3 Project on a finite-dimensional space thanks to **FEM**.

For all test functions  $\vec{v}_q$ ,  $v_p$  and  $v_\partial$  (smooth enough):

$$\begin{cases} \langle \partial_t \vec{\alpha}_q, \vec{v}_q \rangle_{\mathbf{L}^2} = \langle \overrightarrow{\text{grad}}(\mathbf{e}_p), \vec{v}_q \rangle_{\mathbf{L}^2}, \\ \langle \partial_t \alpha_p, v_p \rangle_{L^2} = \langle \text{div}(\vec{\mathbf{e}}_q), v_p \rangle_{L^2}, \\ \langle \mathbf{y}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} = \langle \vec{\mathbf{e}}_q \cdot \vec{\mathbf{n}}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}. \end{cases}$$

Applying Green's formula on the 1st line and using the definition of  $\mathbf{u}$ :

$$\langle \partial_t \vec{\alpha}_q, \vec{v}_q \rangle_{\mathbf{L}^2} = - \langle \mathbf{e}_p, \text{div}(\vec{v}_q) \rangle_{L^2} + \langle \vec{v}_q \cdot \vec{\mathbf{n}}, \mathbf{u} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

The strategy follows:

- 1 Write the **weak formulation**;
- 2 Apply an accurate **Stokes (Green) identity** (such that  $\mathbf{u}$  “appears”);
- 3 Project on a finite-dimensional space thanks to **FEM**.

For all test functions  $\vec{v}_q$ ,  $v_p$  and  $v_\partial$  (smooth enough):

$$\begin{cases} \langle \partial_t \vec{\alpha}_q, \vec{v}_q \rangle_{\mathbf{L}^2} = \langle \overrightarrow{\text{grad}}(\mathbf{e}_p), \vec{v}_q \rangle_{\mathbf{L}^2}, \\ \langle \partial_t \alpha_p, v_p \rangle_{L^2} = \langle \text{div}(\vec{e}_q), v_p \rangle_{L^2}, \\ \langle \mathbf{y}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} = \langle \vec{e}_q \cdot \vec{n}, v_\partial \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}. \end{cases}$$

Applying Green's formula on the 1st line and using the definition of  $\mathbf{u}$ :

$$\langle \partial_t \vec{\alpha}_q, \vec{v}_q \rangle_{\mathbf{L}^2} = - \langle \mathbf{e}_p, \text{div}(\vec{v}_q) \rangle_{L^2} + \langle \vec{v}_q \cdot \vec{n}, \mathbf{u} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

Green's formula applied on the 2nd line would lead to normal stress control  
 $\mathbf{u} = \vec{e}_q \cdot \vec{n}$ . The energy variables are **partitioned** accordingly.

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\vec{\alpha}_q^{ap}(t, \vec{x}) := \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \vec{\alpha}_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t),$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\vec{\alpha}_q^{ap}(t, \vec{x}) := \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \vec{\alpha}_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t), \quad \vec{e}_q^{ap}(t, \vec{x}) = \vec{\Phi}_q^\top \cdot \underline{e}_q(t),$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\begin{aligned}\vec{\alpha}_q^{ap}(t, \vec{x}) &:= \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \underline{\alpha}_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t), & \vec{e}_q^{ap}(t, \vec{x}) &= \vec{\Phi}_q^\top \cdot \underline{e}_q(t), \\ \underline{\alpha}_p^{ap}(t, \vec{x}) &:= \sum_{k=1}^{N_p} \varphi_p^k(\vec{x}) \underline{\alpha}_p^k(t) = \underline{\phi}_p^\top \cdot \underline{\alpha}_p(t), & \underline{e}_p^{ap}(t, \vec{x}) &= \underline{\phi}_p^\top \cdot \underline{e}_p(t),\end{aligned}$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,  $\underline{\phi}_p$  an  $N_p \times 1$  matrix

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\begin{aligned}\vec{\alpha}_q^{ap}(t, \vec{x}) &:= \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \alpha_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t), & \vec{e}_q^{ap}(t, \vec{x}) &= \vec{\Phi}_q^\top \cdot \underline{e}_q(t), \\ \alpha_p^{ap}(t, \vec{x}) &:= \sum_{k=1}^{N_p} \varphi_p^k(\vec{x}) \alpha_p^k(t) = \phi_p^\top \cdot \underline{\alpha}_p(t), & \vec{e}_p^{ap}(t, \vec{x}) &= \phi_p^\top \cdot \underline{e}_p(t), \\ \vec{u}^{ap}(t, \vec{s}) &:= \sum_{m=1}^{N_\partial} \psi^m(\vec{s}) \vec{u}^m(t) = \Psi^\top \cdot \underline{\vec{u}}(t), & \vec{y}^{ap}(t, \vec{s}) &= \Psi^\top \cdot \underline{\vec{y}}(t),\end{aligned}$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,  $\phi_p$  an  $N_p \times 1$  matrix and  $\Psi$  an  $N_\partial \times 1$  matrix.

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\begin{aligned}\vec{\alpha}_q^{ap}(t, \vec{x}) &:= \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \alpha_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t), & \vec{e}_q^{ap}(t, \vec{x}) &= \vec{\Phi}_q^\top \cdot \underline{e}_q(t), \\ \alpha_p^{ap}(t, \vec{x}) &:= \sum_{k=1}^{N_p} \varphi_p^k(\vec{x}) \alpha_p^k(t) = \phi_p^\top \cdot \underline{\alpha}_p(t), & \underline{e}_p^{ap}(t, \vec{x}) &= \phi_p^\top \cdot \underline{e}_p(t), \\ \underline{\mathbf{u}}^{ap}(t, \vec{s}) &:= \sum_{m=1}^{N_\partial} \psi^m(\vec{s}) \underline{\mathbf{u}}^m(t) = \Psi^\top \cdot \underline{\mathbf{u}}(t), & \underline{\mathbf{y}}^{ap}(t, \vec{s}) &= \Psi^\top \cdot \underline{\mathbf{y}}(t),\end{aligned}$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,  $\phi_p$  an  $N_p \times 1$  matrix and  $\Psi$  an  $N_\partial \times 1$  matrix.

The discretized system (giving the structure) then reads:

$$\begin{cases} \vec{M}_q \cdot \frac{d}{dt} \underline{\alpha}_q(t) = D \cdot \underline{e}_p(t) + B \cdot \underline{\mathbf{u}}(t), \\ M_p \cdot \frac{d}{dt} \underline{\alpha}_p(t) = -D^\top \cdot \underline{e}_q(t), \\ M_\partial \cdot \underline{\mathbf{y}}(t) = B^\top \cdot \underline{e}_q(t), \end{cases}$$

# Conservative System: FEM Application

The energy, co-energy, boundary and test functions of the *same* index are discretized by using the *same* bases, either scalar- or **vector**-valued:

$$\begin{aligned}\vec{\alpha}_q^{ap}(t, \vec{x}) &:= \sum_{\ell=1}^{N_q} \vec{\phi}_q^\ell(\vec{x}) \underline{\alpha}_q^\ell(t) = \vec{\Phi}_q^\top \cdot \underline{\alpha}_q(t), & \vec{e}_q^{ap}(t, \vec{x}) &= \vec{\Phi}_q^\top \cdot \underline{e}_q(t), \\ \underline{\alpha}_p^{ap}(t, \vec{x}) &:= \sum_{k=1}^{N_p} \varphi_p^k(\vec{x}) \underline{\alpha}_p^k(t) = \underline{\phi}_p^\top \cdot \underline{\alpha}_p(t), & \underline{e}_p^{ap}(t, \vec{x}) &= \underline{\phi}_p^\top \cdot \underline{e}_p(t), \\ \underline{\mathbf{u}}^{ap}(t, \vec{s}) &:= \sum_{m=1}^{N_\partial} \psi^m(\vec{s}) \underline{\mathbf{u}}^m(t) = \underline{\Psi}^\top \cdot \underline{\mathbf{u}}(t), & \underline{\mathbf{y}}^{ap}(t, \vec{s}) &= \underline{\Psi}^\top \cdot \underline{\mathbf{y}}(t),\end{aligned}$$

with  $\vec{\Phi}_q$  an  $N_q \times 2$  matrix,  $\underline{\phi}_p$  an  $N_p \times 1$  matrix and  $\underline{\Psi}$  an  $N_\partial \times 1$  matrix.

The discretized system (giving the structure) then reads:

$$\begin{cases} \vec{M}_q \cdot \frac{d}{dt} \underline{\alpha}_q(t) = D \cdot \underline{e}_p(t) + B \cdot \underline{\mathbf{u}}(t), \\ M_p \cdot \frac{d}{dt} \underline{\alpha}_p(t) = -D^\top \cdot \underline{e}_q(t), \\ M_\partial \cdot \underline{\mathbf{y}}(t) = B^\top \cdot \underline{e}_q(t), \end{cases}$$

where:

$$\vec{M}_q := \int_{\Omega} \vec{\Phi}_q \cdot \vec{\Phi}_q^\top, \quad M_p := \int_{\Omega} \underline{\phi}_p \cdot \underline{\phi}_p^\top, \quad M_\partial := \int_{\Omega} \underline{\Psi} \cdot \underline{\Psi}^\top,$$

$$D := - \int_{\Omega} \operatorname{div} \left( \vec{\Phi}_q \right) \cdot \underline{\phi}_p^\top, \quad B := \int_{\partial\Omega} \left( \vec{\Phi}_q \cdot \vec{n} \right) \cdot \underline{\Psi}^\top.$$

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

**A** The inner product on  $\mathbb{R}^{N_q}$ ,  $\mathbb{R}^{N_p}$  and  $\mathbb{R}^{N_\partial}$  has to be taken w.r.t. the mass matrices  $\overrightarrow{M}_q$ ,  $M_p$  and  $M_\partial$ : e.g.  $\langle \vec{v}_1, \vec{v}_2 \rangle_{N_q} := \vec{v}_2^\top \cdot \overrightarrow{M}_q \cdot \vec{v}_1$ .

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

⚠ The inner product on  $\mathbb{R}^{N_q}$ ,  $\mathbb{R}^{N_p}$  and  $\mathbb{R}^{N_\partial}$  has to be taken w.r.t. the mass matrices  $\overrightarrow{M}_q$ ,  $M_p$  and  $M_\partial$ : e.g.  $\langle \vec{v}_1, \vec{v}_2 \rangle_{N_q} := \vec{v}_2^\top \cdot \overrightarrow{M}_q \cdot \vec{v}_1$ .

## Discrete Hamiltonian

$$\mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) := \mathcal{H}(\vec{\alpha}_q^{ap}, \alpha_p^{ap}) = \frac{1}{2} \left( \underline{\alpha}_q^\top \cdot \overrightarrow{M}_{\overline{T}} \cdot \underline{\alpha}_q + \underline{\alpha}_p^\top \cdot M_{\frac{1}{\rho}} \cdot \underline{\alpha}_p \right),$$

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

⚠ The inner product on  $\mathbb{R}^{N_q}$ ,  $\mathbb{R}^{N_p}$  and  $\mathbb{R}^{N_\partial}$  has to be taken w.r.t. the mass matrices  $\overrightarrow{M}_q$ ,  $M_p$  and  $M_\partial$ : e.g.  $\langle \vec{v}_1, \vec{v}_2 \rangle_{N_q} := \vec{v}_2^\top \cdot \overrightarrow{M}_q \cdot \vec{v}_1$ .

## Discrete Hamiltonian

$$\begin{aligned} \mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) &:= \mathcal{H}(\vec{\alpha}_q^{ap}, \alpha_p^{ap}) = \frac{1}{2} \left( \underline{\alpha}_q^\top \cdot \overrightarrow{M}_{\overline{\mathbf{T}}} \cdot \underline{\alpha}_q + \underline{\alpha}_p^\top \cdot M_{\frac{1}{p}} \cdot \underline{\alpha}_p \right), \\ \overrightarrow{M}_{\overline{\mathbf{T}}} &:= \int_{\Omega} \vec{\Phi}_q \cdot \overline{\mathbf{T}} \cdot \vec{\Phi}_q^\top \quad \& \quad M_{\frac{1}{p}} := \int_{\Omega} \frac{1}{p} \phi_p \cdot \phi_p^\top. \end{aligned}$$

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

⚠ The inner product on  $\mathbb{R}^{N_q}$ ,  $\mathbb{R}^{N_p}$  and  $\mathbb{R}^{N_\partial}$  has to be taken w.r.t. the mass matrices  $\vec{M}_q$ ,  $M_p$  and  $M_\partial$ : e.g.  $\langle \vec{v}_1, \vec{v}_2 \rangle_{N_q} := \vec{v}_2^\top \cdot \vec{M}_q \cdot \vec{v}_1$ .

## Discrete Hamiltonian

$$\begin{aligned} \mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) &:= \mathcal{H}(\vec{\alpha}_q^{ap}, \alpha_p^{ap}) = \frac{1}{2} \left( \underline{\alpha}_q^\top \cdot \vec{M}_{\bar{\underline{T}}} \cdot \underline{\alpha}_q + \underline{\alpha}_p^\top \cdot M_{\frac{1}{p}} \cdot \underline{\alpha}_p \right), \\ \vec{M}_{\bar{\underline{T}}} &:= \int_\Omega \vec{\Phi}_q \cdot \bar{\underline{T}} \cdot \vec{\Phi}_q^\top \quad \& \quad M_{\frac{1}{p}} := \int_\Omega \frac{1}{p} \phi_p \cdot \phi_p^\top. \end{aligned}$$

**Constitutive relations:**  $\vec{M}_q \cdot \underline{e}_q = \vec{M}_{\bar{\underline{T}}} \cdot \underline{\alpha}_q \quad \& \quad M_p \cdot \underline{e}_p = M_{\frac{1}{p}} \cdot \underline{\alpha}_p \quad \checkmark \checkmark$

## Finite-Dimensional Dirac Structure

$$\mathcal{J}_d := \begin{pmatrix} 0 & D & B \\ -D^\top & 0 & 0 \\ -B^\top & 0 & 0 \end{pmatrix} \implies \mathcal{D}_d := \text{Graph}(\mathcal{J}_d).$$

⚠ The inner product on  $\mathbb{R}^{N_q}$ ,  $\mathbb{R}^{N_p}$  and  $\mathbb{R}^{N_\partial}$  has to be taken w.r.t. the mass matrices  $\vec{M}_q$ ,  $M_p$  and  $M_\partial$ : e.g.  $\langle \vec{v}_1, \vec{v}_2 \rangle_{N_q} := \vec{v}_2^\top \cdot \vec{M}_q \cdot \vec{v}_1$ .

## Discrete Hamiltonian

$$\begin{aligned} \mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) &:= \mathcal{H}(\vec{\alpha}_q^{ap}, \alpha_p^{ap}) = \frac{1}{2} \left( \underline{\alpha}_q^\top \cdot \vec{M}_{\bar{\bar{T}}} \cdot \underline{\alpha}_q + \underline{\alpha}_p^\top \cdot M_{\frac{1}{p}} \cdot \underline{\alpha}_p \right), \\ \vec{M}_{\bar{\bar{T}}} &:= \int_\Omega \vec{\Phi}_q \cdot \bar{\bar{T}} \cdot \vec{\Phi}_q^\top \quad \& \quad M_{\frac{1}{p}} := \int_\Omega \frac{1}{p} \phi_p \cdot \phi_p^\top. \end{aligned}$$

**Constitutive relations:**  $\vec{M}_q \cdot \underline{e}_q = \vec{M}_{\bar{\bar{T}}} \cdot \underline{\alpha}_q$  &  $M_p \cdot \underline{e}_p = M_{\frac{1}{p}} \cdot \underline{\alpha}_p$  ✓✓

Denote  $\underline{f} := \left( \frac{d}{dt} \underline{\alpha}_q, \frac{d}{dt} \underline{\alpha}_p, -\underline{y} \right)^\top$  and  $\underline{e} := \left( \underline{e}_q, \underline{e}_p, \underline{u} \right)^\top$ , then:

## Discrete Lossless Power Balance

$$\begin{pmatrix} \underline{f} \\ \underline{e} \end{pmatrix} \in \mathcal{D}_d \Rightarrow \langle \underline{f}, \underline{e} \rangle_{N_p, N_q, N_\partial} = 0 \Rightarrow \frac{d}{dt} \mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) = \underline{u}^\top \cdot M_\partial \cdot \underline{y}.$$

## 1 Introduction

## 2 Partitioned Finite Element Method (PFEM)

- Conservative System
- **Internal Dissipation**
- Boundary Dissipation

## 3 Conclusion

# Internal Dissipation: Dissipative Ports

The Hamiltonian is always the total energy:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{\mathbf{T}}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

Internal dissipation  $\epsilon(\vec{x}) \partial_t w(t, \vec{x}) = \epsilon(\vec{x}) \vec{e}_p(t, \vec{x})$  is added, with  $\epsilon \geq 0$ :

$$\begin{cases} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(\vec{e}_p), \\ \partial_t \alpha_p = \text{div}(\vec{e}_q) - \epsilon \vec{e}_p, \end{cases} \quad \begin{cases} \vec{u} = \vec{e}_p, \\ \vec{y} = \vec{e}_q \cdot \vec{n}. \end{cases}$$

# Internal Dissipation: Dissipative Ports

The Hamiltonian is always the total energy:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{\mathbf{T}}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

Internal dissipation  $\epsilon(\vec{x}) \partial_t w(t, \vec{x}) = \epsilon(\vec{x}) \vec{e}_p(t, \vec{x})$  is added, with  $\epsilon \geq 0$ :

$$\begin{cases} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(\vec{e}_p), \\ \partial_t \alpha_p = \text{div}(\vec{e}_q) - \epsilon \vec{e}_p, \end{cases} \quad \begin{cases} \mathbf{u} = \vec{e}_p, \\ \mathbf{y} = \vec{e}_q \cdot \vec{n}. \end{cases}$$

$$\begin{pmatrix} \partial_t \vec{\alpha}_q \\ \partial_t \alpha_p \end{pmatrix} = \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & -\epsilon \end{pmatrix} \begin{pmatrix} \vec{e}_q \\ \vec{e}_p \end{pmatrix} \rightsquigarrow J := \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & 0 \end{pmatrix}, \quad R := \begin{pmatrix} 0 & 0 \\ 0 & \epsilon \end{pmatrix}.$$

# Internal Dissipation: Dissipative Ports

The Hamiltonian is always the total energy:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{\mathbf{T}}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

Internal dissipation  $\epsilon(\vec{x}) \partial_t w(t, \vec{x}) = \epsilon(\vec{x}) \mathbf{e}_p(t, \vec{x})$  is added, with  $\epsilon \geq 0$ :

$$\begin{cases} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(\mathbf{e}_p), \\ \partial_t \alpha_p = \text{div}(\vec{\mathbf{e}}_q) - \epsilon \mathbf{e}_p, \end{cases} \quad \begin{cases} \mathbf{u} = \mathbf{e}_p, \\ \mathbf{y} = \vec{\mathbf{e}}_q \cdot \vec{\mathbf{n}}. \end{cases}$$

$$\begin{pmatrix} \partial_t \vec{\alpha}_q \\ \partial_t \alpha_p \end{pmatrix} = \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & -\epsilon \end{pmatrix} \begin{pmatrix} \vec{\mathbf{e}}_q \\ \mathbf{e}_p \end{pmatrix} \rightsquigarrow J := \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & 0 \end{pmatrix}, \quad R := \begin{pmatrix} 0 & 0 \\ 0 & \epsilon \end{pmatrix}.$$

Adding **dissipative ports**  $\mathbf{f}_r$  and  $\mathbf{e}_r$  and a **dissipative constitutive relation**:

$$\oplus \mathbf{e}_r = \epsilon \mathbf{f}_r \quad \begin{pmatrix} \partial_t \vec{\alpha}_q \\ \partial_t \alpha_p \\ \mathbf{f}_r \end{pmatrix} = \begin{pmatrix} 0 & \overrightarrow{\text{grad}} & 0 \\ \text{div} & 0 & -I \\ 0 & I & 0 \end{pmatrix} \begin{pmatrix} \vec{\mathbf{e}}_q \\ \mathbf{e}_p \\ \mathbf{e}_r \end{pmatrix}.$$

# Internal Dissipation: Dissipative Ports

The Hamiltonian is always the total energy:

$$\mathcal{H}(\vec{\alpha}_q, \alpha_p) := \frac{1}{2} \int_{\Omega} \left( \vec{\alpha}_q \cdot \bar{\bar{\mathbf{T}}} \cdot \vec{\alpha}_q + \frac{1}{\rho} \alpha_p^2 \right).$$

Internal dissipation  $\epsilon(\vec{x}) \partial_t w(t, \vec{x}) = \epsilon(\vec{x}) \mathbf{e}_p(t, \vec{x})$  is added, with  $\epsilon \geq 0$ :

$$\begin{cases} \partial_t \vec{\alpha}_q = \overrightarrow{\text{grad}}(\mathbf{e}_p), \\ \partial_t \alpha_p = \text{div}(\vec{\mathbf{e}}_q) - \epsilon \mathbf{e}_p, \end{cases} \quad \begin{cases} \mathbf{u} = \mathbf{e}_p, \\ \mathbf{y} = \vec{\mathbf{e}}_q \cdot \vec{\mathbf{n}}. \end{cases}$$

$$\begin{pmatrix} \partial_t \vec{\alpha}_q \\ \partial_t \alpha_p \end{pmatrix} = \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & -\epsilon \end{pmatrix} \begin{pmatrix} \vec{\mathbf{e}}_q \\ \mathbf{e}_p \end{pmatrix} \rightsquigarrow J := \begin{pmatrix} 0 & \overrightarrow{\text{grad}} \\ \text{div} & 0 \end{pmatrix}, \quad R := \begin{pmatrix} 0 & 0 \\ 0 & \epsilon \end{pmatrix}.$$

Adding **dissipative ports**  $\mathbf{f}_r$  and  $\mathbf{e}_r$  and a **dissipative constitutive relation**:

$$\oplus \xrightarrow{\mathbf{e}_r = \epsilon \mathbf{f}_r} \begin{pmatrix} \partial_t \vec{\alpha}_q \\ \partial_t \alpha_p \\ \mathbf{f}_r \end{pmatrix} = \begin{pmatrix} 0 & \overrightarrow{\text{grad}} & 0 \\ \text{div} & 0 & -I \\ 0 & I & 0 \end{pmatrix} \begin{pmatrix} \vec{\mathbf{e}}_q \\ \mathbf{e}_p \\ \mathbf{e}_r \end{pmatrix}.$$

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}_q, \alpha_p) = - \langle \epsilon \mathbf{e}_p, \mathbf{e}_p \rangle_{L^2} + \langle \mathbf{y}, \mathbf{u} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} \leq \langle \mathbf{y}, \mathbf{u} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

# Internal Dissipation: PFEM

Approximating  $\underline{f}_r$  and  $\underline{e}_r$  in the FEM basis  $\phi_p$ , PFEM gives:

$$\underbrace{\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 \\ 0 & 0 & M_p & 0 \\ 0 & 0 & 0 & M_\partial \end{pmatrix}}_{\mathcal{M}} \underbrace{\begin{pmatrix} \frac{d}{dt} \underline{\alpha}_q(t) \\ \frac{d}{dt} \underline{\alpha}_p(t) \\ \underline{f}_r(t) \\ -\underline{y}(t) \end{pmatrix}}_{\vec{f}_d} = \underbrace{\begin{pmatrix} 0 & D & 0 & B \\ -D^\top & 0 & M_p & 0 \\ 0 & -M_p & 0 & 0 \\ -B^\top & 0 & 0 & 0 \end{pmatrix}}_{\mathcal{J}_d} \underbrace{\begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{u}(t) \end{pmatrix}}_{\vec{e}_d}.$$

# Internal Dissipation: PFEM

Approximating  $\underline{f}_r$  and  $\underline{e}_r$  in the FEM basis  $\phi_p$ , PFEM gives:

$$\underbrace{\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 \\ 0 & 0 & M_p & 0 \\ 0 & 0 & 0 & M_\partial \end{pmatrix}}_{\mathcal{M}} \underbrace{\begin{pmatrix} \frac{d}{dt} \underline{\alpha}_q(t) \\ \frac{d}{dt} \underline{\alpha}_p(t) \\ \underline{f}_r(t) \\ -\underline{y}(t) \end{pmatrix}}_{\vec{f}_d} = \underbrace{\begin{pmatrix} 0 & D & 0 & B \\ -D^\top & 0 & M_p & 0 \\ 0 & -M_p & 0 & 0 \\ -B^\top & 0 & 0 & 0 \end{pmatrix}}_{\mathcal{J}_d} \underbrace{\begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{u}(t) \end{pmatrix}}_{\vec{e}_d}.$$

The **dissipative constitutive relation** is discretized as:

$$M_p \cdot \underline{e}_r = \underline{E} \cdot \underline{f}_r, \quad \text{with } \underline{E} := \int_{\Omega} \epsilon \phi_p \cdot \phi_p^\top \geq 0.$$

# Internal Dissipation: PFEM

Approximating  $\underline{f}_r$  and  $\underline{e}_r$  in the FEM basis  $\phi_p$ , PFEM gives:

$$\underbrace{\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 \\ 0 & 0 & M_p & 0 \\ 0 & 0 & 0 & M_\partial \end{pmatrix}}_{\mathcal{M}} \underbrace{\begin{pmatrix} \frac{d}{dt} \underline{\alpha}_q(t) \\ \frac{d}{dt} \underline{\alpha}_p(t) \\ \underline{f}_r(t) \\ -\underline{y}(t) \end{pmatrix}}_{\vec{f}_d} = \underbrace{\begin{pmatrix} 0 & D & 0 & B \\ -D^\top & 0 & M_p & 0 \\ 0 & -M_p & 0 & 0 \\ -B^\top & 0 & 0 & 0 \end{pmatrix}}_{\mathcal{J}_d} \underbrace{\begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{u}(t) \end{pmatrix}}_{\vec{e}_d}.$$

The **dissipative constitutive relation** is discretized as:

$$M_p \cdot \underline{e}_r = \underline{E} \cdot \underline{f}_r, \quad \text{with } \underline{E} := \int_{\Omega} \epsilon \phi_p \cdot \phi_p^\top \geq 0.$$

The **extended Dirac structure**  $\mathcal{D}_d^\epsilon := \text{Graph}(\mathcal{J}_d)$ , w.r.t. the  $\mathcal{M}$ -weighted scalar product in  $\mathbb{R}^{N_q + 2N_p + N_\partial}$ , takes into account for any  $\epsilon \geq 0$ .

# Internal Dissipation: PFEM

Approximating  $f_r$  and  $e_r$  in the FEM basis  $\phi_p$ , PFEM gives:

$$\underbrace{\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 \\ 0 & 0 & M_p & 0 \\ 0 & 0 & 0 & M_\partial \end{pmatrix}}_{\mathcal{M}} \underbrace{\begin{pmatrix} \frac{d}{dt} \underline{\alpha}_q(t) \\ \frac{d}{dt} \underline{\alpha}_p(t) \\ \underline{f}_r(t) \\ -\underline{y}(t) \end{pmatrix}}_{\vec{f}_d} = \underbrace{\begin{pmatrix} 0 & D & 0 & B \\ -D^\top & 0 & M_p & 0 \\ 0 & -M_p & 0 & 0 \\ -B^\top & 0 & 0 & 0 \end{pmatrix}}_{\mathcal{J}_d} \underbrace{\begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{u}(t) \end{pmatrix}}_{\vec{e}_d}.$$

The **dissipative constitutive relation** is discretized as:

$$M_p \cdot \underline{e}_r = \underline{E} \cdot \underline{f}_r, \quad \text{with } \underline{E} := \int_{\Omega} \epsilon \phi_p \cdot \phi_p^\top \geq 0.$$

The **extended Dirac structure**  $\mathcal{D}_d^\epsilon := \text{Graph}(\mathcal{J}_d)$ , w.r.t. the  $\mathcal{M}$ -weighted scalar product in  $\mathbb{R}^{N_q+2N_p+N_\partial}$ , takes into account for any  $\epsilon \geq 0$ .

## Discrete Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}_d (\underline{\alpha}_q, \underline{\alpha}_p) = -\underline{e}_p^\top \cdot \underline{E} \cdot \underline{e}_p + \underline{u}^\top \cdot M_\partial \cdot \underline{y} \leq \underline{u}^\top \cdot M_\partial \cdot \underline{y}.$$

⚠ In practice,  $f_r$  and  $e_r$  do not need to be discretized in the basis of  $f_p$  and  $e_p$ .

## 1 Introduction

## 2 Partitioned Finite Element Method (PFEM)

- Conservative System
- Internal Dissipation
- Boundary Dissipation

## 3 Conclusion

# Boundary Dissipation: Impedance Ports

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\vec{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

# Boundary Dissipation: Impedance Ports

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\vec{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

**A** This kind of dissipation does not easily fit in the “ $J - R$  framework”.

# Boundary Dissipation: Impedance Ports

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\vec{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

**A** This kind of dissipation does not easily fit in the “ $J - R$  framework”.  
It can be seen as an *output feedback law*  $u = -Zy + \nu$  in the previous case.

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\bar{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

⚠ This kind of dissipation does not easily fit in the “ $J - R$  framework”.  
It can be seen as an *output feedback law*  $u = -Zy + \nu$  in the previous case.

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}_q, \alpha_p) = -\langle \epsilon e_p, e_p \rangle_{L^2} - \langle y, Z y \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} + \langle y, \nu \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

# Boundary Dissipation: Impedance Ports

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\bar{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

⚠ This kind of dissipation does not easily fit in the “ $J - R$  framework”.

It can be seen as an *output feedback law*  $u = -Zy + \nu$  in the previous case.

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}_q, \alpha_p) = -\langle \epsilon e_p, e_p \rangle_{L^2} - \langle y, Z y \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} + \langle y, \nu \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

Add **impedance ports** ( $f_i, e_i$ ) and **dissipative constitutive relation**  $e_i = Z f_i$ , and approximate  $f_i$  and  $e_i$  in the FEM basis  $\Psi$ , PFEM gives:

$$\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 & 0 \\ 0 & 0 & M_p & 0 & 0 \\ 0 & 0 & 0 & M_\partial & 0 \\ 0 & 0 & 0 & 0 & M_\partial \end{pmatrix} \begin{pmatrix} \frac{d}{dt} \alpha_q(t) \\ \frac{d}{dt} \alpha_p(t) \\ \underline{f}_r(t) \\ \underline{f}_i(t) \\ -\underline{y}(t) \end{pmatrix} = \begin{pmatrix} 0 & D & 0 & -B & B \\ -D^\top & 0 & M_p & 0 & 0 \\ 0 & -M_p & 0 & 0 & 0 \\ B^\top & 0 & 0 & 0 & 0 \\ -B^\top & 0 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{e}_i(t) \\ \underline{\nu}(t) \end{pmatrix}$$

and  $M_\partial \cdot \underline{e}_i = \langle Z \rangle \cdot \underline{f}_i$ , with  $\langle Z \rangle := \int_{\partial\Omega} Z \Psi \cdot \Psi^\top \geq 0$ .

# Boundary Dissipation: Impedance Ports

The Impedance Boundary Condition (IBC), with  $Z \geq 0$  on  $\partial\Omega$ , and  $\nu$  as new control, is considered:  $\nu = e_p + Z \vec{e}_q \cdot \vec{n} \Leftrightarrow \nu = \partial_t w + Z (\bar{\bar{T}} \cdot \vec{\text{grad}}(w)) \cdot \vec{n}$ .

⚠ This kind of dissipation does not easily fit in the “ $J - R$  framework”.

It can be seen as an *output feedback law*  $u = -Zy + \nu$  in the previous case.

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}(\vec{\alpha}_q, \alpha_p) = -\langle \epsilon e_p, e_p \rangle_{L^2} - \langle \mathbf{y}, Z \mathbf{y} \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}} + \langle \mathbf{y}, \nu \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

Add **impedance ports**  $(f_i, e_i)$  and **dissipative constitutive relation**  $e_i = Z f_i$ , and approximate  $f_i$  and  $e_i$  in the FEM basis  $\Psi$ , PFEM gives:

$$\begin{pmatrix} \vec{M}_q & 0 & 0 & 0 & 0 \\ 0 & M_p & 0 & 0 & 0 \\ 0 & 0 & M_p & 0 & 0 \\ 0 & 0 & 0 & M_\partial & 0 \\ 0 & 0 & 0 & 0 & M_\partial \end{pmatrix} \begin{pmatrix} \frac{d}{dt} \alpha_q(t) \\ \frac{d}{dt} \alpha_p(t) \\ \underline{f}_r(t) \\ \underline{f}_i(t) \\ \underline{\mathbf{y}}(t) \end{pmatrix} = \begin{pmatrix} 0 & D & 0 & -B & B \\ -D^\top & 0 & M_p & 0 & 0 \\ 0 & -M_p & 0 & 0 & 0 \\ B^\top & 0 & 0 & 0 & 0 \\ -B^\top & 0 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} \underline{e}_q(t) \\ \underline{e}_p(t) \\ \underline{e}_r(t) \\ \underline{e}_i(t) \\ \underline{\nu}(t) \end{pmatrix}$$

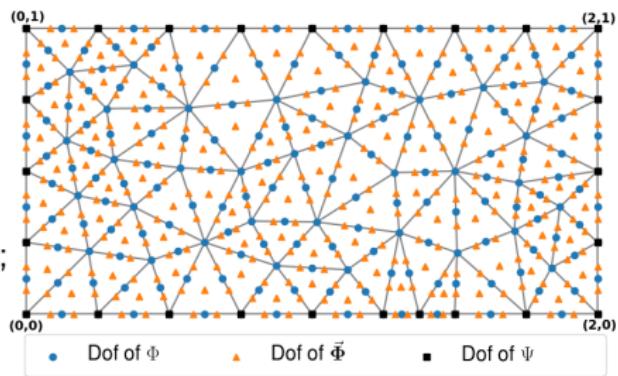
$$\text{and } M_\partial \cdot \underline{e}_i = \langle Z \rangle \cdot \underline{f}_i, \quad \text{with } \langle Z \rangle := \int_{\partial\Omega} Z \Psi \cdot \Psi^\top \geq 0.$$

## Discrete Lossy Power Balance

$$\frac{d}{dt} \mathcal{H}_d(\underline{\alpha}_q, \underline{\alpha}_p) = -\underline{e}_p^\top \cdot E \cdot \underline{e}_p - \underline{\mathbf{y}}^\top \cdot \langle Z \rangle \cdot \underline{\mathbf{y}} + \underline{\nu}^\top \cdot M_\partial \cdot \underline{\mathbf{y}}.$$

# Boundary Dissipation: Simulations

- Heterogenous ( $\rho \not\equiv \text{constant}$ );
- Anisotropic (tensor  $\bar{\mathbf{T}} \not\equiv \text{constant}$ );
- $\epsilon \equiv 0$ ;
- $Z \neq 0$  for  $t \geq 2$ ;
- Raviart-Thomas FEM for  $q$ -variables;
- Lagrange FEM for  $p$ -variables;
- Lagrange FEM for  $\partial$ -variables;



1 Introduction

2 Partitioned Finite Element Method (PFEM)

3 Conclusion

## To sum up:

A structure-preserving method has been proposed for dissipative port-Hamiltonian Systems, with the following strategy:

- **Add ports** to get a Dirac structure;
- Write down **weak formulations**;
- Apply **Stokes formula on a Partition** of the system;
- Apply the **Finite Elements Method**;

**Furthermore:** *diffusion model* as **heat equation** can be handled.

## To go further:

- Choice for the **finite elements families**:
  - Convergence rate?
  - *Conformity*:  $\mathcal{D}_d \subset \mathcal{D}$ ?
- **Mixed** boundary control;
- **Symplectic** time-integration?  *DAE!!!*

-  A structure-preserving Partitioned Finite Element Method for the 2D wave equation  
**Cardoso-Ribeiro F.L., Matignon D., Lefèvre L.**  
*IFAC-PapersOnLine, vol.51(3), pp.119–124 (2018)*, LHMNC 2018
-  Structure preserving approximation of dissipative evolution problems  
**Egger H.**  
*Numerische Mathematik, vol.143(1), pp.85–106 (2019)*
-  Energy-Preserving and Passivity-Consistent Numerical Discretization of Port-Hamiltonian Systems  
**Celledoni E., Høiseth, E.H.**  
*arXiv:1706.08621, (2017)*
-  Hamiltonian formulation of distributed-parameter systems with boundary energy flow  
**van der Schaft A. J., Maschke B.**  
*Journal of Geometry and Physics, vol.42(1–2), pp.166–194 (2002)*

Thank you for your attention!

## ■ Space domain and physical parameters:

- $\Omega \subset \mathbb{R}^{n \geq 1}$  is a bounded open connected set;
- $\vec{n}$  is the outward unit normal on the boundary  $\partial\Omega$ ;
- $\rho(\vec{x})$  is the mass density;
- $\overline{\overline{T}}(\vec{x})$  is the conductivity tensor.

## ■ Notations:

- $T$  is the local temperature;
- $\beta := \frac{1}{T}$  is the reciprocal temperature;
- $u$  is the internal energy density;
- $s$  is the entropy density;
- $\vec{J}_Q$  is the heat flux;
- $\vec{J}_S := \beta \vec{J}_Q$  is the entropy flux;
- $C_V := \left( \frac{du}{dT} \right)_V$  is the isochoric heat capacity.

## ■ “Context & Axioms”:

- **Medium:** rigid body without chemical reaction;
- **1st law of thermodynamics:**

$$\rho(\vec{x})\partial_t u(t, \vec{x}) = -\operatorname{div}(\vec{J}_Q(t, \vec{x}));$$

## ■ Gibbs' relation:

$$dU = T dS, \implies \partial_t u(t, \vec{x}) = T(t, \vec{x})\partial_t s(t, \vec{x});$$

## ■ Entropy evolution:

$$\rho(\vec{x})\partial_t s(t, \vec{x}) = -\operatorname{div}(\vec{J}_S(t, \vec{x})) + \sigma(t, \vec{x}),$$

with  $\sigma := \overrightarrow{\operatorname{grad}}(\beta) \cdot \vec{J}_Q$  is the *irreversible entropy production*.

## ■ “Laws”:

### ■ Fourier's law:

$$\vec{J}_Q(t, \vec{x}) = -\overline{\bar{T}}(t, \vec{x}) \cdot \overrightarrow{\operatorname{grad}}(T(t, \vec{x}));$$

### ■ Dulong-Petit's law:

$$u(t, \vec{x}) = C_V(\vec{x})T(t, \vec{x}).$$

## Quadratic Hamiltonian: Lyapunov Functional

$$\mathcal{H}(u(t, \vec{x})) := \frac{1}{2} \int_{\Omega} \rho(\vec{x}) \frac{(u(t, \vec{x}))^2}{C_V(t, \vec{x})} \, d\vec{x},$$

$\alpha_u := u$  is the **energy variable**, and  $e_u := \delta_{\alpha_u}^{\rho} = \frac{u}{C_V}$  the **co-energy variable**.

Under *Dulong-Petit's law*, this is the *usual* functional used in the mathematics community:  $\mathcal{H} := \int_{\Omega} \rho C_v T^2$ , even if **its physical meaning is far to be clear**.

## Power Balance

$$\frac{d}{dt} \mathcal{H} = \int_{\Omega} \vec{J}_Q \cdot \overrightarrow{\text{grad}} \left( \frac{u}{C_V} \right) - \int_{\partial\Omega} \frac{u}{C_V} \vec{J}_Q \cdot \vec{n} - \frac{1}{2} \int_{\Omega} \rho \partial_t C_V \left( \frac{u}{C_V} \right)^2.$$

Defining  $f_u := \partial_t \alpha_u = \partial_t u$ ,  $e_u = \frac{u}{C_V}$ ,  $\vec{f}_Q := -\overrightarrow{\text{grad}} \left( \frac{u}{C_V} \right)$ , and  $\vec{e}_Q := \vec{J}_Q$ :

$$\begin{pmatrix} \rho \vec{f}_u \\ \vec{f}_Q \end{pmatrix} = \begin{pmatrix} 0 & -\text{div} \\ -\overrightarrow{\text{grad}} & 0 \end{pmatrix} \begin{pmatrix} e_u \\ \vec{e}_Q \end{pmatrix}.$$

# Diffusion: Lyapunov Functional

At least two choices for **boundary control**:  $\underline{e}_u$  or  $\overrightarrow{e}_Q \cdot \overrightarrow{n}$ .

With **inward flux control**  $\nu = -\overrightarrow{e}_Q \cdot \overrightarrow{n}$ , the output is  $\underline{y} = \overrightarrow{e}_u$ , i.e. the **boundary temperature** using Dulong-Petit's law, and the discretized system is:

$$\begin{pmatrix} M_{\rho} & 0 & 0 \\ 0 & \overrightarrow{M} & 0 \\ 0 & 0 & M_{\partial} \end{pmatrix} \begin{pmatrix} \underline{f}_u \\ \underline{f}_Q \\ -\underline{y} \end{pmatrix} = \begin{pmatrix} 0 & D & B \\ -D^T & 0 & 0 \\ -B^T & 0 & 0 \end{pmatrix} \begin{pmatrix} \underline{e}_u \\ \underline{e}_Q \\ \underline{\nu} \end{pmatrix},$$

& **constitutive relations**:  $M_{\rho C_V} \cdot \frac{d}{dt} \underline{e}_u = M_{\rho} \cdot \underline{f}_u$  &  $\overrightarrow{M} \cdot \overrightarrow{e}_Q = \overrightarrow{M}_{\overline{T}} \cdot \underline{f}_Q$ .

## Lossy Power Balance

$$\frac{d}{dt} \mathcal{H} := - \int_{\Omega} \overrightarrow{f}_Q \cdot \overline{\overline{T}} \cdot \overrightarrow{f}_Q + \int_{\partial\Omega} \underline{y} \nu.$$

## Discrete Lossy Power Balance

$$\frac{d}{dt} \mathcal{H} := - \underline{f}_Q \cdot \overrightarrow{M}_{\overline{T}} \cdot \underline{f}_Q + \underline{\nu}^T \cdot M_{\partial} \cdot \underline{y}.$$

Let us take as Hamiltonian the internal energy in function of the entropy:

$$\mathcal{U}(s(t, \vec{x})) := \int_{\Omega} \rho(\vec{x}) u(s(t, \vec{x})) \, d\vec{x},$$

together with  $\nu = T$  and  $\mathbf{y} = \vec{J}_S \cdot \vec{n}$ .

**Power Balance** (first law of thermodynamics)

$$\frac{d}{dt} \mathcal{U}(s) = \langle \mathbf{y}, \nu \rangle_{H^{-\frac{1}{2}}, H^{\frac{1}{2}}}.$$

Adding *entropy ports* with the **entropy constitutive relation** (definition of  $\sigma$ ):  $T\sigma = -\overrightarrow{\text{grad}}(T) \cdot \vec{J}_S$ , leads to a PHDAE.

*Gibbs' relation is a first constitutive relation, and Fourier's law can be the other.*

**Discrete Power Balance**

$$\frac{d}{dt} \mathcal{U}_d(\bar{s}) = \nu^\perp \cdot M_\partial \cdot \mathbf{y}.$$



**Institut Supérieur de l'Aéronautique et de l'Espace**

10 avenue Édouard Belin – BP 54032

31055 Toulouse Cedex 4 – France

Phone: +33 5 61 33 80 80

[www.isae-supatra.fr](http://www.isae-supatra.fr)